Successes and Challenges of Incompressible Flow Simulation

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ABSTRACT

During the past thirty years, numerical methods and simulation tools for incompressible flows have been advanced as a subset of CFD discipline. Even though incompressible flows encountered in many areas of engineering, simulation of compressible flow has been the major driver for developing computational algorithms and tools. This is probably due to rather stringent requirements for predicting aerodynamic performance characteristics of flight vehicles, while flow devices involving low speed or incompressible flow could be reasonably well designed without resorting to accurate numerical simulations. As flow devices are required to be more sophisticated and highly efficient, CFD tools become indispensable in fluid engineering for incompressible and low speed flow. This paper is intended to review some of the successes made bv advances in computational technologies during the same period, and discuss some of the current challenges.

I. INTRODUCTION

Almost all, if not all, fluid is compressible. Incompressible flow is an approximation of flow where flow speed is insignificant everywhere compared to the speed of sound of the medium. Mathematical expressions of incompressible flow, such as the incompressible Navier-Stokes equations, are therefore approximations to the real world. If incompressible flow is defined this way, the majority of the fluid and associated flow we encounter in our daily lives belong to the incompressible category. For example, most of the flow associated with air and water, (such as flow related to automobile, ships, submarines, water supply through pipes and channels, hydraulic turbine, pumps, low speed airplane and

trout swimming in mountain stream) and flow of biofluid (such as blood) are all in the incompressible flow domain. One of the earliest mathematical models of incompressible flow is the famous equation by Bernoulli, who developed the model equation while investigating blood flow in 1730 (see an interesting article by Quarteroni [1]). It is not surprising to find many attempts to investigate incompressible flow both analytically, experimentally and computationally ever since.

Computational study of incompressible flow problems both in basic research and in industrial applications has been performed for several decades. Numerical solutions for such basic fluid dynamics problems as flow past a circular cylinder, flow through channel, duct and pipe, flow over a backward facing step were presented as early as 1930s (e.g. Thom [2] for circular cylinder). In the computational fluid dynamics (CFD) community, especially in aerospace, CFD synonymous computational has been to aerodynamics. Computational incompressible fluid dynamics was treated as a special case of aerodynamics, that is, the low speed limit of compressible fluid dynamics. This is probably due to rather stringent requirements for predicting aerodynamic performance characteristics of flight vehicles, while flow devices involving low speed flow can be reasonably well designed without resorting to accurate numerical simulations (e.g. hydraulic turbine for hydroelectric power plants and liquid fuel and oxidizer devices for the Space Shuttle main engine were designed without help from CFD approach). As flow devices become increasingly compact and highly efficient, pushing the conventional operating envelope, requirements on incompressible CFD tools became more demanding just as aerodynamic performance prediction tools require quantitative prediction capability. This trend is reflected into

the development of various incompressible flow solution methods and associated tools, especially, in conjunction with high-fidelity computations using high-end computing facilities.

Mathematically. the incompressible flow formulation poses unique issues not present in compressible equations because of the incompressibility requirement. Physically, information travels at infinite speed in an incompressible medium, which imposes stringent requirement to computational algorithm of satisfying incompressibility as well as difficulties in designing downstream boundary conditions. Differences in various methods of solving the incompressible flow equations originate from different strategies of satisfying incompressibility.

There are generally two different approaches in solving the incompressible flow equations, i.e. incompressible Navier-Stokes equations. The first one is based on satisfying the incompressibility directly. If primitive variables, pressure and velocity, are chosen, pressure is used as a mapping parameter to satisfy incompressibility. This class of methods is generally known as "pressure-based" method. Instead of pressure and velocity, derived quantities like stream functionvorticity and vorticity-velocity can be used resulting in different sets of governing equations. general three-dimensional applications, however, the primitive variable formulation poses the least difficulty in geometry modeling and in setting the boundary conditions. The second approach is based on compressible flow formulation where momentum and continuity equations are coupled through the use of density. Incompressibility is recovered as a limiting case of this formulation. This class of methods is known as "density-based" method. The artificial compressibility method is a typical of this which is a special case approach. preconditioned compressible flow methods.

During the last several years, a large number of review articles and books on CFD discussed incompressible flow methods. For more comprehensive reviews of computational methods for incompressible flow in general, readers are referred to these materials, i.e. Roach [3], Peyret and Taylor [4], Hirsch [5], Kwak [6], Gunzburger and Nicolades [7], Hafez and Oshima [8], Gresho and Sani [9] and Hafez [10]. In these books and articles, formulations, numerical methods and solutions to fundamental fluid dynamic problems are presented fairly extensively. In the present

paper, a short summary of methods from a historical perspective is given. After thirty some years of numerical simulation of incompressible flows, industrial problems involving complex systems are now solved rather routinely. The current review is focused on these applications to answer questions like what have been accomplished to date and what challenges still remain to be resolved. There are a vast number of these cases, and those presented here are, of course, only limited samples to illustrate the author's opinion.

II. EVOLUTION OF SOLUTION METHODS

For general applications, the formulation based on primitive variables is the most flexible choice. Other formulations using derived variables, such as stream function and vorticity, eliminate the need for pressure boundary conditions, however, at the expense of adding a new requirement for derived variables such as vorticity boundary conditions. All these formulations produce comparable results for two-dimensional problems. However, extension to three dimensions is not straightforward for stream function formulation. For vorticity-velocity formulation in three dimensions, three Poisson-type vorticity equations are introduced that makes the formulation very expensive to solve. In the present paper, examples of three-dimensional applications are primarily obtained using primitive variable formulation. Therefore, in this section, evolution of the two most popular approaches using primitive variables is briefly reviewed. Other formulations and examples can be found in the literature [3-10].

2.1 Formulation

Three-dimensional incompressible flow with constant density is governed by the following Navier-Stokes equations:

$$\frac{\partial u_i}{\partial x_i} = 0 \tag{2.1}$$

$$\frac{\partial u_i}{\partial t} + \frac{\partial u_i u_j}{\partial x_i} = -\frac{\partial p}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_i} \quad (2.2)$$

Where t is the time, x_i the Cartesian coordinates, u_i , the corresponding velocity components, p the pressure, and τ_{ij} the viscous stress tensor. Here, all variables are non-dimensionalized by a

reference velocity and length scale. For turbulent flow, τ_{ii} should include turbulent stresses.

The major difference between the incompressible and the compressible Navier-Stokes formulation is in the continuity equation. The incompressible formulation can be viewed as a singular limit of the compressible one. Satisfying the mass conservation equation, therefore, is the primary issue in solving the above set of equations. Physically, incompressible flow is characterized by elliptic behavior of the pressure waves, the speed of which in a truly incompressible flow is infinite. The pressure field is desired as a part of the solution. However, the pressure condition is the boundary for needed on numerical computation. One can utilize other formulation using derived quantities like vorticity to eliminate the pressure from the boundary conditions; however, boundary conditions for the derived variables have to be imposed. In realistic 3-D problems, these derived quantities are difficult to define or impractical to use. The primitive variable formulation, namely, using pressure and velocities as dependent variables, then becomes more convenient and flexible in 3-D applications. However, in this formulation, mass conservation and its relation to pressure has to be properly handled while achieving computational efficiency. Various techniques have been developed in the past, none of which have been proven to be universally better than the others.

2.2 Methods Based on Pressure Iteration

In 1965, Harlow and Welch [11] published the first primitive variable method using a derived Poisson equation for pressure. In this method, called the marker-and-cell (MAC) method, the pressure is used as a mapping parameter to satisfy the continuity equation. By taking the divergence of the momentum equation, the Poisson equation for pressure is obtained:

$$\nabla^2 p = \frac{\partial h_i}{\partial x_i} - \frac{\partial}{\partial t} \frac{\partial u_i}{\partial x_i} = g \qquad (2.3)$$

Where

$$h_i = -\frac{\partial u_i u_j}{\partial x_i} + \frac{\partial \tau_{ij}}{\partial x_j}$$

The usual computational procedure involves choosing the pressure field at the current time step such that continuity is satisfied at the next time step. The original MAC method is based on a staggered arrangement on a 2-D Cartesian grid. The staggered grid conserves mass, momentum, and kinetic energy in a natural way and avoids odd-even point decoupling of the pressure encountered in a regular grid [12]. However, these properties become unclear when applied to generalized curvilinear coordinates. Even though the original method used an explicit Euler solver, various time advancing schemes can be implemented here. Ever since its introduction, numerous variations of the MAC method have been devised and successful computations have heen made.

The major drawback of this method is the large amount of computing time required for solving the Poisson equation for pressure. When the physical problem requires a very small time step, the penalty paid for an iterative solution procedure for the pressure may be tolerable. But the method as a whole is slow and the pressure boundary condition is difficult to specify. One important aspect of the numerical solution of the Poisson equation for pressure is tied to the spatial differencing of the second derivatives. To satisfy the mass conservation in grid space, the difference form of the second derivative in the Poisson equation has to be constructed consistently with the discretized momentum equation (see Kwak [6] for more detail).

Since, for a steady-state solution, the correct pressure field is desired only when the solution is converged, the iteration procedure for the pressure can be simplified such that it requires only a few iteration at each time step. The best-known method using this approach is the Semi-Implicit Method for Pressure-Linked Equations (SIMPLE) [13-14]. The unique feature of this method is the simple way of estimating the velocity correction and thus the pressure correction. This feature simplifies the computation but introduces empiricism into the method. Despite its empiricism, the method has been used successfully for many computations. This method has been the basis for many pressure-based methods developed ever since (e.g. [15] and many others). It is not, however, the intention of this paper to fully review the evolution of this method. Many successful applications and many tools developed to date are probably the real testimonial of the value of this pressure-based approach in general.

Fractional-step procedure for pressure-based method

The MAC method can be viewed as a special case of the projection method (i.e. Chorin, [16]) or, from the operator point of a view, one variation of the fractional step method [17-19]). Here, operator splitting can be accomplished in several ways by treating the momentum equation as a combination of convection, pressure, and viscous terms. The common application of this method is done in two steps. The first step is to solve for an auxiliary velocity field using the momentum equation in which the pressure-gradient term can be computed from the pressure in the previous time step or can be excluded entirely. In the second step, the pressure is computed which can map the auxiliary velocity onto a divergence-free velocity field. This procedure was first applied in staggered Cartesian grid by Harlow and Welch [11], and later extended to regular grids (e.g. Dwyer [20] and many others).

One particular aspect of the fractional step method requiring special care is the intermediate boundary condition. Orszag et al. [21] discussed this extensively. Rosenfeld et al. [22] devised a generalized scheme where physical boundary conditions can be used at intermediate steps. As with other pressure-based methods, the efficiency of the fractional step method depends on the Poisson solver. A multi-grid acceleration, which is physically consistent with the elliptic field, is one possible avenue to enhance the computational efficiency. In light of successful computations in Cartesian coordinates using staggered grid, it became of interest to develop a method for general three-dimensional coordinates. However, extension of the staggered arrangement to general computational cells is not straightforward and involves features related to conservation. A general numerical method based on a staggered grid was developed at NASA Ames among others, initially by Rosenfeld et al. [22]. Later, Kiris and Kwak [23] developed a more robust implicit procedure for not so "nice grid" using finite-volume framework on a staggered grid. A staggered grid has favorable properties in Cartesian coordinates, such as coupling odd-even points. However, it is debatable whether a staggered grid has clear advantages over a regular grid in generalized curvilinear coordinates. In practice, this is because that flux balancing in general curved cells requires interpolation that makes the differences between cell-centered and staggered arrangement

insignificant. Further discussion related to this approach can be found in recent collection of papers by Hafez [10] (for example, in papers by Loner et. al [24] and more).

2.3 Artificial Compressibility Method

Advances in CFD in aerospace have been made in conjunction with compressible flow computations. Therefore, it is of significant interest to be able to use some of these compressible flow algorithms. One of the early attempts to do this is the artificial compressibility method by Chorin (1967) [25]. In this formulation, the continuity equation is modified by adding a time-derivative of the pressure term resulting in

$$\frac{1}{\beta} \frac{\partial p}{\partial t} + \frac{\partial u_i}{\partial x_i} = 0 \tag{2.4}$$

where β is an artificial compressibility or a pseudo-compressibility parameter. The two terms, artificial compressibility and pseudo-compressibility, are used interchangeably in the literature. Together with the unsteady momentum equations, this forms a hyperbolic-parabolic type of time-dependent system of equations. Thus, the implicit schemes developed for compressible flows can be implemented. It is to be noted that t no longer represents a true physical time in this formulation.

Physically, this means that waves of finite speed are introduced into the incompressible flow field as a medium to distribute the pressure. truly incompressible flow, the wave speed is infinite, whereas the speed of propagation of these pseudo waves depends on the magnitude of the artificial compressibility. In a true incompressible flow, the pressure field is affected instantaneously by any disturbances in the flow field, but with artificial compressibility, there will be a time lag between the flow disturbance and its effect on the pressure field. Ideally, the value of the artificial compressibility is chosen to be as high as the particular choice of algorithm allows, so that the incompressibility is recovered quickly. This has to be done without lessening the accuracy and the stability property of the numerical method implemented. On the other hand, if the artificial compressibility is chosen such that these waves travel too slowly, then the variation of the pressure field accompanying these waves is very This will interfere with the timely development of the viscous boundary layer. In viscous flows, the behavior of the boundary layer

is very sensitive to the streamwise pressure gradient, especially when the boundary layer is separated. If separation is present, a pressure wave traveling with finite speed will cause a change in the local pressure gradient that will affect the location of the flow separation. This change in separated flow will feed back to the pressure field, possibly preventing convergence to a steady state. Especially for internal flow, the viscous effect is important for the entire flow field and the interaction between the pseudo pressure-waves and the viscous flow field becomes very important.

Artificial compressibility relaxes the strict requirement of satisfying mass conservation in each step. However, to utilize this convenient feature, it is essential to understand the nature of artificial compressibility both physically and mathematically. Chang and Kwak (1984) [26] reported details of artificial compressibility, and suggested some useful guidelines for choosing the artificial compressibility parameter.

For obtaining steady state solutions, the governing equations are iterated in pseudo time until the solution converges to steady state, at which time original incompressible Navier-Stokes are satisfied. Time dependent equations calculation of incompressible flows are especially time consuming due to the elliptic nature of the governing equations. Physically, this means that any local change in the flow has to be felt by the entire flow field. Numerically, this means that in each time step, the pressure field has to go through one complete steady-state iteration cycle, for example, by Poisson-solver-type pressure iteration or artificial compressibility iteration method. In transient flow, the physical time step has to be small and consequently the change in the flow field may be small. In this situation, the number of iterations in each time step for getting a divergence-free flow field may not be as high as regular steady-state computations. However, time-accurate computations are in general extremely time-consuming. Therefore, it is particularly desirable to develop computationally efficient methods by implementing a fast algorithm and by utilizing computer characteristics such as parallel processing.

Various applications, which evolved from this concept, have been reported for obtaining steady-state solutions (e.g. Kwak et al. [27], Chang et al. [28], Choi and Merkle [29]). To obtain time-dependent solutions using this method, an

iterative procedure can be applied in each physical time step such that the continuity equation is satisfied. Merkle and Athavale [30] and Rogers et. al [31] reported successful computations using this pseudo time iteration approach.

Pre-conditioning

In incompressible flow, the density is constant, which is the source of difficulty in solving the compressible Navier-Stokes equation as the flow approaches the incompressible limit. Adding a non-physical pressure term made it possible to adopt a time-like iteration scheme in solving the governing equations. The success of this approach prompted a series of attempts to develop preconditioning methods to be used in conjunction with compressible Navier-Stokes equations (Turkel [32], Van Leer et. al [33], Choi and Merkle [34], Venkataswaran and Merkle [35]). This method is especially valuable for solving aerodynamic flow field involving multiple scales (i.e. flow field involving regions of very low speed in generally high speed flow problems). This formulation is also being utilized in simulating multi-phase flow such as cavitation in liquid flow [36].

Numerical Schemes

It is easy to see that the artificial compressibility code resembles that of a compressible flow code. For example, to alleviate the difficulty of solving an unfactored implicit scheme, indirect methods have been devised by many researchers. The alternating direction implicit scheme by Beam and Warming [37] or Briley and McDonald [38] approximates the implicit operator in the unfactored scheme by a product of three one-dimensional operators. This scheme is unconditionally stable in two dimensions. In three dimensions, with the use of numerical dissipation, this scheme becomes conditionally stable. The ADI scheme introduces factorization error that reduces the rate of convergence. However, this method introduced perhaps the first implicit scheme to CFD, thus viscous flow computation became more feasible. With the help of reduced cost by the diagonalization of the Jacobian by Pulliam and Chaussee [39], this scheme has been used in the development of many flow solvers. At NASA Ames, the first incompressible Navier-Stokes solver in generalized three-dimensional coordinate, INS3D code (Kwak et. al [27]), was developed using this scheme. The original INS3D code was extensively used as a primary design tool for upgraded design of the Space Shuttle main engine (SSME) hot gas

manifold by Chang et. al [28]. This was probably the first major contribution by CFD in rocket propulsion system design.

In artificial compressible formulation, since the governing equations are changed into the hyperbolic-parabolic type, some of the upwind differencing schemes, which have been developed for the compressible Euler and Navier-Stokes equations [40-43] can be used. These schemes were implemented into the flow solver successfully using artificial compressibility method by Rogers et. al [31]. A variety of iterative schemes can be implemented in conjunction with the upwind schemes. For example, Rogers et. al [31] solved the resulting set of numerical equations using a unfactored line relaxation scheme similar to that employed by MacCormack [44].

2.4 Other Methods

Even though the evolution of incompressible flow solver development was explained using examples the present authors are directly involved with, a large number of research effort has been on-going in the incompressible CFD community in general. Especially, development along finite element / or unstructured grid approach has not been touched in this paper, and readers interested in this area are referred to a rather recent book by Gresho and Sani [9], and a number of articles in the book by Hafez [10] (for example, see articles by Glowinski [45], Morgan [46], Tezduyar [47] and a large number of articles covering numerical methods and special topics). Also, a number of other methods related to incompressible flow computations can be found in the references cited.

In general it is the authors opinion that there are no major differences any more in basic methodologies and tools used among different organizations. Differences in robustness, computational efficiencies and accuracy usually come from how the solution procedures are set up in conjunction with geometry and grid handling, software engineering and some subtle algorithmic aspects not very well presentable in written papers.

3. Computed Examples

Solutions to some basic problems

Validation computations of solution algorithms and flow solvers have been reported in the

literature over the years, e.g., problems like square cavity, flow over a circular cylinder, duct flow of various shapes, backward facing steps, juncture flow and tip and wake vortex flow. Even though the geometry of these flow problems are relatively simple, they offer rich sources of studying flow physics and opportunities to prove the validity of particular numerical methods and procedures. One simple example is shown in Figure 1 where artificial compressibility and pressure projection methods are compared to evaluate the capability for predicting unsteady phenomena. Once the final solutions are obtained, the two approaches can produce equivalent results, however, the iterative processes for the artificial compressibility method and time advancing scheme for the pressure projection scheme had to be properly handled [48].

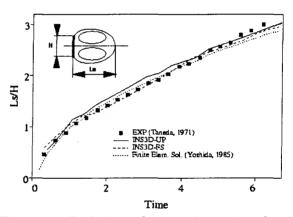


Figure 1. Evolution of stagnation point for an impulsively started flat plate: Comparison of artificial compressibility (denoted by INS3D-UP), projection method (INS3D-FS), finite element solution [49] and water tank experiment [50].

Validation for real world applications

Validation of solution procedure for real-world applications takes another level of effort because the flow field involving complex geometry often requires high resolution locally to resolve the complicated flow as well as requiring evaluation of physical model being implemented. Such validation effort is of significant importance to problems like submarine with propulsor and turbopump subsystems for liquid rocket engine.

Figure 2 illustrates one such validation using a pump impeller. This validation is the first step for further application of the simulation procedure to an entire turbopump system. Figure 2a shows impeller geometry with surface grid where test is conducted

for developing an advanced impeller. Figure 2b shows some comparison of computed results with test data. Even with this simplified geometry, testing is not straight forward, which involves variations in flow conditions and shroud geometry. Simulation involves geometric modeling and grid generation as well as approximation related to numerical methods and physical modeling. As shown, the computed results are not on top of test data unlike the basic flow problems cited above. In engineering applications, especially, for preliminary design applications, one needs to determine what level of accuracy is acceptable. More importantly the numerical simulation procedure must produce consistent results when flow conditions are varied as well as for variations in geometry such as different angle, length and lean of impeller blades.

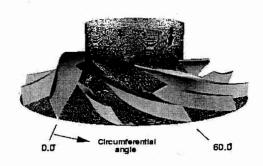


Figure 2a. Geometry of a test impeller.

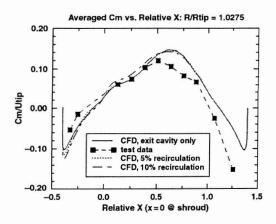


Figure 2b. Comparison of circumferentially averaged meridional velocity at the impeller exit.

High-end computing for large engineering problems

One of the most significant applications of incompressible flow simulation started in the early 1980s, when NASA carried out a series of

upgrades to the Space Shuttle Main Engine (SSME) developed in the 1970s. One such effort was powerhead redesign. The powerhead, considered the backbone of the engine, consists of the main injector assembly and pre-burners. Partially burned hot gas passes through Hot Gas Manifold (HGM) to the main injector assembly. During 1983-84, the powerhead redesign was undertaken, focusing on two-duct HGM. NASA Ames and Rocketdyne collaborated in applying the INS3D code for this task. Since the Mach number of the partially burned gas was very low, the incompressible approximation was a valid assumption. The team of researchers successfully applied CFD simulation procedure to this task for enhancing the performance of the SSME powerhead. The new design was developed based on two-duct hot gas manifold. The two-duct design replaced the previous three-duct engine, smoothing the fuel flow, reducing pressure and turbulence, and lowering temperatures in the engine during operation. This two-duct design first flew on the shuttle in July 1995. It significantly improved fluid flow in the system, thus reducing maintenance and enhancing the overall performance of the engine. This pioneering work was probably the first major application of incompressible CFD to rocket propulsion system (see [28] for more detail).

A large number of incompressible flow software has been developed ever since for analyzing complex engineering systems. One such example is the UNCLE code [36], which has been applied to various naval hydrodynamic applications. There are, however, many other equally significant software in existence. It is not the intention of this report to list the inventory of such codes.

A typical process of flow simulation, especially, high-fidelity unsteady flow, is illustrated next using a turbopump similar to the one used in the Shuttle engine (figure 3).

The goal is to develop a computational framework for the design and analysis of an entire liquid rocket engine fuel supply system, which is of major importance in developing a future rocket engine to be used in space launch. A substantial computational time reduction for 3D unsteady flow simulations is required to reduce the design-cycle time of the pumps. Part of this speed up will be due to enhancements in computer hardware. The remaining portion of the speed-up must be contributed by advances in grid generation

procedure, flow solution algorithms and by efficient parallel implementations. In the present example, an overset strategy [51] was adopted to generate the grid. To reduce human and computing time for generating multiples of pump component, scripting [52] has been developed thus substantially reducing the grid generation portion of the entire simulation time. Figure 4 illustrate this procedure.

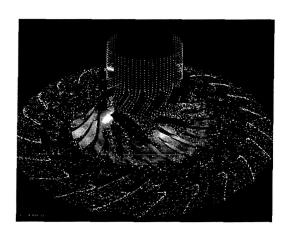


Figure 3. A typical turbopump for a rocket engine showing the geometry and unsteady solution.

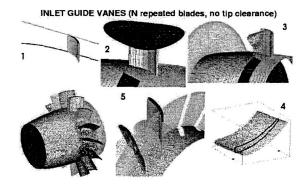


Figure 4. Illustration of automated procedure with turbopump scripts for inlet guide vanes.

Parallel computing strategies vary depending on computer architecture such as memory arrangement relative to processing units. Two approaches have been implemented in the INS3D framework: the first approach is hybrid MPI/OpenMP and the second one is Multi-Level Parallelism (MLP) developed at NASA-Ames Research Center. The first approach is obtained by using message-passing interface (MPI) for inter-zone parallelism, and by using OpenMP directives for intra-zone parallelism. INS3D-MPI

is based on the explicit message-passing interface across MPI groups and is designed for coarse grain parallelism.

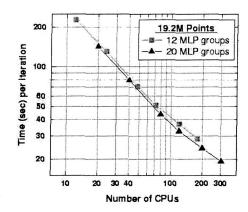


Figure 5. Parallel performance of INS3D-MLP on SGI Origin platform.

The second approach in Multi-Level Parallelism (MLP) is obtained by using NAS-MLP routines. This approach differs from the MPI/OpenMP approach in a fundamental way in that it does not use messaging at all. All data communication at the coarsest and finest level is accomplished via direct memory referencing instructions. This approach also provides a simpler mechanism for converting legacy code, such as INS3D, than MPI. For shared memory MLP, the coarsest level parallelism is supplied by spawning off independent processes via the standard UNIX fork. The boundary data for the overset grid system is archived in the shared memory arena by each process. Other processes access the data from the arena as needed. Figure 5 shows the MLP scalability for SSME impeller computations using 19.2 Million grid points.

Biomedical / Biofluid Applications

Extension of incompressible flow methods to blood flow has been of interest to biomedical researchers for many years. For the cardiovascular system, the brain and other parts of the human body, branching of blood vessels involve bifurcations, most of which are non-symmetric. Therefore, bifurcation has been a popular subject for blood flow simulations. Bifurcation problems offer the opportunity to address numerical issues involving grid generation as well as to study basic fluid dynamic phenomena relevant to blood circulation simulation. When the size of blood vessel becomes very small as in capillaries, non-Newtonian characteristics become significant,

thus requiring physical modeling. In addition the vessel wall is elastic requiring a structural model to account for geometric change depending on wall stresses. Early pioneering work on this subject was done by Womersley [53]. Later more sophisticated model and computations have been performed for regular and non-regular bifurcating arteries including stenotic vessels (see Berger [54] for a review).

The human circulatory system is like a huge tree with many branches of various different sizes. Therefore, a lot of computational study has been performed using a truncated geometric model. One difficulty of simulating a truncated arterial system comes from setting proper boundary conditions, especially at the downstream boundary. To account for large arterial network, Quarteroni and his colleagues have a developed circuit analog, which has been applied to more inclusive circulatory systems modeling such as the cardiovascular system and the circle of Willis in the brain (see [55] among many publications). An alternative way of imposing downstream boundary conditions has developed by Tim David and his colleagues [56] in their computation of the circle of Willis.

Modeling and simulation of cardiovascular system has been the subject of many investigations because of its importance in biomedical research. Peter Hunter and his colleagues have been modeling cardiovascular systems including multidisciplinary aspects producing one of the most impressive results to date (e.g. see [57]). Earlier and perhaps the most elaborate physical model of the heart has been pioneered by Peskin and his colleagues (see [58] for example). They included blood, wall structures and electrical field activating heart muscles.

Another interesting application of blood flow simulation is related to artificial devices such as artificial heart, ventricular assist device (VAD), and heart valves. Since the demand for organs for transplantation far exceeds the number of donors, the need for artificial devices become increasingly high, to be used either as a temporary device or as a permanent replacement for natural organ. For developing these devices, accurate quantification of blood flow plays a crucial role. Thus CFD simulation of blood flow in and around these artificial devices has become an indispensable part of the design. One such example is the recent effort of CFD-aided design of DeBakey VAD, which enabled human implantation by removing

thrombus formation and lowering hemolysis to an acceptable level for human applications. This effort set a new milestone for CFD applications to the biomedical area [59].

All such blood flow computations may be regarded as a branch of incompressible CFD. The number of CFD applications to blood flow and biomedical problems are increasing rapidly, and the work cited here represents only small samples of the vast amount of work existing and on-going.

4. Challenges

Incompressible CFD methodology has been advanced along with computational technologies in general. Many fluid engineering problems could now be simulated, however, mostly at a single problem level. For example, it is possible to generate solutions to problems like a turbopump pump (inducer-impeller-diffuser), a naval vehicle at a steady motion including propulsor by a model, and a truncated model of the brain or heart. To realize the full benefit of CFD, more inclusive modeling is desired, such as, systems of pump including multiples of pumps and feed line, vehicle in maneuver with propulsor, complete or more inclusive human circulatory system from heart through aorta all the way to brain and kidney capillaries. Attempts to solve these have been made with some qualitative success. However, the predictive capability is still very limited and prediction with accurate physics is yet be accomplished, which will require inclusion of not only fluid dynamic modeling but modeling of other quantities like thermal loading, structural properties (such as structural behavior of arterial wall), and cavitation physics. These computations will require not only large computing resources but also large data storage and management technologies.

Flow solver codes and software tools have been developed to the point that many daily fluid engineering problems can now be routinely computed. Some of the physical models such as for turbulence and transition, however, have not been advanced much more than what were available in the 70s or 80s. Other models like cavitation model are yet to be advanced to produce quantitative results for engineering. In aerospace design, the most productive aspect of CFD applications has been to predict relative change among design variations. To push the limit of operation and to try bold new ideas, more predictive capability will be needed for

complicated flows involving transient phenomena, separation, tip vortex and cavitation. For example, without accurate prediction capability for such quantities like cavitation and damaging frequency, back flow and rotational stall; CFD cannot be of much help to the development of an advance turbopump. To make these advances, high-fidelity computation using high-end computing facilities will still be a "must" despite all the euphoria about the PC clusters and grid computing.

One final note is related to human resources. Even though CFD has been advanced remarkably, many challenging cases require CFD experts. Computer

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science can automate a good portion of the CFD simulation processes thus saving a large amount of human time required. However, blind application of tools without understanding capabilities and limitations of the methods involved could lead to catastrophic engineering results. Like in many other engineering and science disciplines, CFD researchers and practitioners need to understand the physics and engineering systems being simulated. Future experts will need to be produced who are willing to think through the flow physics in addition to software engineering.

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